

A-LEVEL Mathematics

MM04 Mechanics 4 Mark scheme

6360

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Key to mark scheme abbreviations

Μ	mark is for method
m or dM	mark is dependent on one or more M marks and is for method
А	mark is dependent on M or m marks and is for accuracy
В	mark is independent of M or m marks and is for method and
	accuracy
E	mark is for explanation
\checkmark or ft or F	follow through from previous incorrect result
CAO	correct answer only
CSO	correct solution only
AWFW	anything which falls within
AWRT	anything which rounds to
ACF	any correct form
AG	answer given
SC	special case
OE	or equivalent
A2,1	2 or 1 (or 0) accuracy marks
–x EE	deduct x marks for each error
NMS	no method shown
PI	possibly implied
SCA	substantially correct approach
С	candidate
sf	significant figure(s)
dp	decimal place(s)

No Method Shown

Where the question specifically requires a particular method to be used, we must usually see evidence of use of this method for any marks to be awarded.

Where the answer can be reasonably obtained without showing working and it is very unlikely that the correct answer can be obtained by using an incorrect method, we must award **full marks**. However, the obvious penalty to candidates showing no working is that incorrect answers, however close, earn **no marks**.

Where a question asks the candidate to state or write down a result, no method need be shown for full marks.

Where the permitted calculator has functions which reasonably allow the solution of the question directly, the correct answer without working earns **full marks**, unless it is given to less than the degree of accuracy accepted in the mark scheme, when it gains **no marks**.

Otherwise we require evidence of a correct method for any marks to be awarded.

Q	Solution	Mark	Total	Comment
1	$\operatorname{Tan} 60^{0} = \frac{h/2}{r}$ $h = 2\sqrt{3}r$	M1 A1 A1	3	Use of tangent to form an equation or suitable moments taken Fully correct equation Accept decimal equivalent $h = 3.46r$
	Total		3	

Q	Solution	Mark	Total	Comment
2 (a)	Take moments about A	marit	Total	
- (~)	Pl = 50(3l)	M1		Use of moments for complete system –
	P = 150 N	A1	2	CAO
(b)(i)	Balancing forces for the whole system Vertical component at $A = 50$ N (upwards) Horizontal component at $A = 150$ N (left)			
	Magnitude = $\sqrt{150^2 + 50^2} = 50\sqrt{10}$ N	M1		Balances system and uses Pythagoras theorem
	Hence $k = 50$	A1	2	Correct <i>k</i> value obtained – implied by correct magnitude
(ii)		B1	1	Correct direction clearly shown
(c)	Rods <i>CD</i> , <i>ED</i> and <i>BE</i> are in compression	B1	1	All correct – no extras
(d)	Resolve vertically at C T _{CD} sin θ = 50	M1		Resolves correctly at C at least once 1 2
	$T_{CD}=50\sqrt{5} N$ (AWRT 112N)	A1		and uses $\sin\theta = \frac{1}{\sqrt{5}}$ or $\cos\theta = \frac{1}{\sqrt{5}}$ or $\theta = 26.6^{\circ}$
	Resolves horizontally at C $T_{BC} = T_{CD} \cos\theta$ $T_{BC}=100 N$	A1F		Resolves correctly and uses their answer for T_{CD}
	Resolve vertically at E $T_{BE}\cos 45^0 = T_{AE}$ Resolving vertically at A $T_{AE} = 50$			
	$I_{AE} = JU$	M1		Resolves correctly and uses sufficient
	$T_{BE} = 50\sqrt{2} N$ (AWRT 71N)	A1	5	equations to find T_{BE}
	Total		11	
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Q	Solution	Mark	Total	Comment
3 (a)	$\int \dots dv = \int_{a}^{a} dv^{2} \dots v^{2} dv$			
	$\int y dx = \int_{-a}^{a} dx - x dx$			
	$a \begin{bmatrix} r^3 \end{bmatrix}$	M1		Applies correct formula to find area and
	$= \left a^2 x - \frac{x}{3} \right $			integrates – one term correct
	$-a \square \square \square$			
	$=\frac{4a}{2}$	A1		Substitutes correct limits to obtain correct
	5			answer
	$\frac{1}{2}\int y^2 dx = \frac{1}{2}\int_{-a}^{a} (a^2 - x^2)^2 dx$			
	$-\frac{1}{a}\int_{a}^{a}(a^{4}-2a^{2}x^{2}+x^{4})dx$			
	$-2\int_{-a}^{a} (a - 2a + x) dx$	MIT		Applies correct formula, expands and integrates – one term correct
	$\begin{bmatrix} 1 & a \end{bmatrix} \begin{bmatrix} a & 2a^2x^3 & x^5 \end{bmatrix}$			
	$-\frac{1}{2} \begin{bmatrix} a & x - \frac{1}{3} + \frac{1}{5} \end{bmatrix}$	A1		Integrates correctly – all terms
	o ⁵	M1		Substitutes correct limits
	$=\frac{\delta a}{15}$			
	15			
	Hence $\overline{K} = \frac{8a^5}{15} / 2a^2$	M1		Forms centre of mass equation
	Hence $I = \frac{4a^3}{5} = \frac{4a^3}{5}$			
	Coordinates are $\left(0^{\frac{2a^2}{2a}}\right)$			
	$\left(0, \frac{1}{5}\right)$	A1	7	Coordinates must be stated correctly
(b)	Takes moments about point A			
	$(2ab)(\frac{b}{2})(2\rho) = (\frac{4a^3}{2a^2})(\rho)$	M1		Forms moment equation
		Α1 Δ1		Correct use of area x distance - rectangle
		M1		Correct use of density ratio applied to their
	$h = 2\sqrt{15}$			equation
	$v = \frac{15}{15}a$	A1		Rearranged - any equivalent form
			5	
	Total		12	

Q	Solution	Mark	Total	Comment
4 (a)	Force \mathbf{F}_3 has zero moment about origin			
	Moment of couple = $\mathbf{r}_1 \times \mathbf{F}_1 + \mathbf{r}_2 \times \mathbf{F}_2$	M1		Forms correct sum of moments
	$\mathbf{r}_{1} \ge \mathbf{F}_{1} = \begin{pmatrix} 2\\1\\-1 \end{pmatrix} \times \begin{pmatrix} -2\\0\\1 \end{pmatrix} = \begin{pmatrix} 1\\0\\2 \end{pmatrix}$	A1		CAO
	$\mathbf{r}_{2} \ge \mathbf{F}_{2} = \begin{pmatrix} 3 \\ -2 \\ 0 \end{pmatrix} \times \begin{pmatrix} 0 \\ 4 \\ -2 \end{pmatrix} = \begin{pmatrix} 4 \\ 6 \\ 12 \end{pmatrix}$	A1		САО
	Total moment of couple = $\begin{pmatrix} 5 \\ 6 \\ 14 \end{pmatrix}$	A1F	4	Correct total of their two non-zero moments
(b)	System in equilibrium so resultant force = 0 Hence $\mathbf{F}_3 = -(\mathbf{F}_1 + \mathbf{F}_2)$	M1		Sets up equation to find F_3
	$\mathbf{F}_3 = \begin{pmatrix} -4 \\ 1 \end{pmatrix}$	A1		Obtains correct F ₃
	Let F ₃ act through point (x, y, z) Moment of F ₃ about origin =			
	$\begin{pmatrix} x \\ y \\ z \\ z$	M1		Finds moment of their \mathbf{F}_3 about origin
	(z) (1) (-4x-2y)	A1		All components correct
	Total moment of all forces must be zero to be in equilibrium hence $\begin{pmatrix} y+4z \\ 2z-x \\ -4x-2y \end{pmatrix} = \begin{pmatrix} -5 \\ -6 \\ -14 \end{pmatrix}$	М1		Sets up equation to ensure zero total moment
	By inspection x = 0 y = 7 z = -3	M1 A1		Solves their equations Any correct point found
	A possible vector equation is $\mathbf{r} = \begin{pmatrix} 0\\7\\-3 \end{pmatrix} + t \begin{pmatrix} 2\\-4\\1 \end{pmatrix}$	M1 A1F	9	Correct structure used – their point and \mathbf{F}_3 used Fully correct – LHS and RHS – follow through their values Possible answers – for any constant a $\mathbf{r} = \begin{pmatrix} 2a \\ 7-4a \\ -3+a \end{pmatrix} + t \begin{pmatrix} 2 \\ -4 \\ 1 \end{pmatrix}$ or

		$\mathbf{r} = \begin{pmatrix} 6+2c\\-5-4c\\c \end{pmatrix} + t \begin{pmatrix} 2\\-4\\1 \end{pmatrix}$ for any constant <i>c</i>
Total	13	

Q	Solution	Mark	Total	Comment
5 (a)	$I_{G} = \frac{1}{3}m \Big[(2a)^{2} + (1.5a)^{2} \Big] = \frac{25}{12}ma^{2}$	B1		I _G correctly obtained
	Using parallel axis theorem $I_A = I_G + md^2 = \frac{25ma^2}{12} + m[(1.5a)^2 + (2a)^2]$	M1		Use of I_G and correct d
	$=\frac{25ma^2}{3}$	A1F	3	Follow through incorrect I _G
(b)(i)	Gain in KE = $\frac{1}{2}(\frac{25ma^2}{3})\dot{\theta}^2 = \frac{25ma^2}{6}\dot{\theta}^2$	B1F		Correct KE – follow through (a)
	Loss in PE = $\frac{5}{2}mga\sin\theta$	B1		Correct PE
	Using gain in KE = loss in PE			
	$\frac{25ma^2}{6}\dot{\theta}^2 = \frac{5}{2}mga\sin\theta$	M1		Forms energy equation
	Hence $\dot{\theta}^2 = \frac{3g\sin\theta}{5a}$	A1	4	CSO - AG
(ii)	Differentiating gives			
	$2\ddot{\theta}\ddot{\theta} = \frac{3g\cos\theta\dot{\theta}}{5a}$	M1		Differentiates both sides – must differentiate
	$\ddot{\theta} = \frac{3g\cos\theta}{10a}$	A1	2	Correct answer obtained
(iii)	Newton's law along AG gives			

	$Y - mg\sin\theta = \frac{5ma}{2}\theta^2$	M1A1		M1 – one side correct A1 both correct
	$Y = mg\sin\theta + \frac{5ma}{2}\left(\frac{3g\sin\theta}{5a}\right) = \frac{5mg\sin\theta}{2}$	A1		Substitutes given expression from 9(b)(i) to obtain force along AG
	Newton's law perpendicular to AG $mg \cos \theta - X = \frac{5ma}{2}\ddot{\theta}$	M1A1		M1 – one side correct A1 both correct
	$X = mg\cos\theta - \frac{5ma}{2} \left(\frac{3g\cos\theta}{10a}\right) = \frac{mg\cos\theta}{4}$	A1F		Substitutes their expression from 9(b)(ii) to obtain force perpendicular to AG
	Resultant force = $\sqrt{\left(\frac{mg\cos\theta}{4}\right)^2 + \left(\frac{5mg\sin\theta}{2}\right)^2}$	M1		Use of Pythagoras for resultant force – dependent on two M1 s above
	$= \frac{mg}{4} \sqrt{\cos^2 \theta + 100 \sin^2 \theta}$ $= \frac{mg}{4} \sqrt{(1 - \sin^2 \theta) + 100 \sin^2 \theta}$ $= \frac{mg}{4} \sqrt{1 + 99 \sin^2 \theta}$	A1F	8	Correct simplification and use of trig identity to obtain result – follow through their (b)(ii)
(iv)	$\frac{5mg}{2}$	B1F	1	Follow through part (iii)
	Total		18	

Q	Solution	Mark	Total	Comment
6 (a)	$\rho = \frac{M}{M}$	B1		ρ and <i>M</i> linked and used anywhere
	$r \pi r^2$			
	Mass of elemental 'hoop' = $2\pi\rho\partial xx$	M1		Considers elemental hoop - mass correct
	MI of each hoop = $2\pi\rho\partial xx^3$	A1		Use of Mr^2 with elemental hoop
	MI disc= $\int_{0}^{r} 2\pi \rho x^{3} dx = \int_{0}^{r} 2\frac{M}{r^{2}} x^{3} dx$	M1		Integrates – integrand must be of correct form
	$= \int_{0}^{r} \left[\frac{2Mx^4}{4r^2} \right] = \frac{Mr^2}{2}$	A1	5	CSO – AG
(b)(i)	$6mg - T_2 = 6mr\ddot{\theta}$	M1		Forms a correct acceleration equation
	$T_1 - 3mg = 3mr\ddot{\theta}$	A1		Both equations correct and $r \ddot{\theta}$ used
	Ratio gives $3T_2 = 4T_1$			
	$3(6 \operatorname{mg} - 6 \operatorname{mr} \ddot{\theta}) = 4(3 \operatorname{mg} + 3 \operatorname{mr} \ddot{\theta})$	M1		Use of tension ratio to reduce to a single equation
	$30mr\ddot{\theta} = 6mg$			
	$\ddot{\Theta} = \frac{g}{5r}$	A1	4	CSO
(ii)	Using part (b)(i)			
	$T_1 = \frac{18mg}{5}$	B1F		Obtains their expression for T ₁
	$T_2 = \frac{24mg}{5}$ For pulley	B1F		Obtains their expression for T ₂
	$T_2 r - T_1 r = I \ddot{\theta}$ $6mg \qquad x g$	M1		Forms correct pulley equation
	$\frac{-5}{5}r = \frac{15}{5r}$			
	$\mathbf{I} = 6mr^2$	M1		Substitutes $T_2, T_1, \ddot{\theta}$ and makes comparison with standard result to obtain
	MI for disc = $\frac{1}{2}Mr^2$			mass
	Mass of pulley = $12m$	A1	5	
(iii)				Correct mass obtained - CAO

Hence $\theta = \frac{3r\omega^2}{g}$ Total		4 18	ensure negative sign is dealt with correctly
Using laws of constant angular acceleration $0^{2} - \omega^{2} = 2\left(-\frac{g}{6r}\right)\theta$	M1 A1		Forms equation to find the angle required
Using C = I $\ddot{\theta}$ Gives $-mgr = 6mr^2 \ddot{\theta}$ $\ddot{\theta} = -\frac{g}{6r}$	M1 A1		Forms equation to find new acceleration Correct acceleration found